



## **ACTRIS recommendation for aerosol inlets and sampling tubes**

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### **Aerosol sampling**

An ideal aerosol sampling system allows an undisturbed sample flow from the environment right to the instrumentation, meanwhile removing some unwanted air ingredients, such as hydrometeors or excessive moisture.

An ideal sampling system

- excludes precipitation and fog droplets from the sampled aerosol,
- provides a representative ambient aerosol sample under as little diffusional and inertial losses,
- provides aerosol particles at a low relative humidity (< 40%),
- minimizes the evaporation of volatile particulate species.

The most common set-ups combine an outdoor aerosol inlet, smooth transport pipes, an aerosol conditioner to dry the sampling flow, and a final flow splitter to distribute the aerosol among the various instruments and samplers. Aerosol instrumentation should generally be housed in a room that provides a clean laboratory environment and temperatures between 15 and 30°C. Optimum indoor temperatures range between 20 and 25 °C.

### **Size cut-offs**

The cut-off size of the aerosol inlet and the height above ground are usually guided by the purpose of the measurement network. The most widely used options are currently PM<sub>10</sub>, PM<sub>2.5</sub>, or PM<sub>1</sub>, implying upper aerodynamic cut-off diameters at 10, 2.5, and 1 µm, respectively, under ambient conditions. These inlets are based on particle separation by either an impactor or a cyclone.

Observational networks, such as WMO-GAW, recommend an upper cut point of 10 µm at ambient conditions (WMO-GAW report 153). The rationale is that particles larger than 10 µm tend to be of local origin and are, thus, not representative for the regional-scale aerosol and its impact on climate effects. TSP (Total Suspended Matter) inlets, in contrast, turn out to be sensitive towards wind speed and cannot provide representative samples of larger particles. To obtain additional sizing information, aerodynamic size cuts 2.5 µm (ambient conditions) and 1 µm (dry conditions) are recommended by WMO-GAW to distinguish fine and coarse particles. The recommendations of the WMO-GAW report 153 were also adopted by EMEP and the European Infrastructure Projects EUSAAR and ACTRIS.

### Whole-air inlet for extreme ambient conditions

Alternative inlet designs might be considered for measurements in an extreme climate. Sampling sites that experience frequent clouds, fog or freezing may prefer using a heated whole-air inlet to capture cloud and fog droplets within the sample. This inlet concerns sites which are located in Polar regions or on high Alpine mountains. Figure 1 illustrates the concept of such a heated whole-air inlet based on the design of the inlet of the Jungfrauoch station in Switzerland as described in Weingartner et al. (1999). Heating prevents clogging of the inlet with ice. Inside the inlet, cloud and fog droplets are evaporated, so that all aerosol particles, whether activated or not, will be included in the measurement. For such whole-air inlets it is desirable to scrutinize the relationship between the ambient wind velocity and variations in the size-cut characteristics.

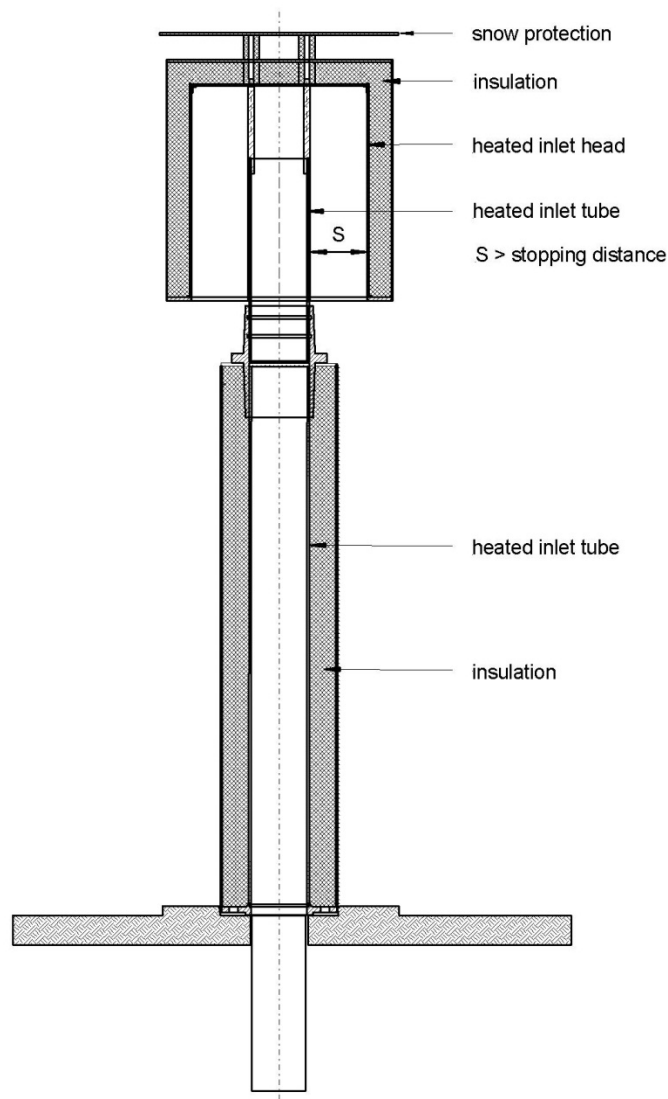


Figure 1: sketch of a whole air inlet

### Tubing and flow splitters

Inside the measurements station, the aerosol flow is usually distributed among several instruments. For aerosol particles, care should be taken with the choice of the tubing and the design of flow distribution devices. Pipes conducting aerosol should be manufactured from metal, preferably stainless steel. It is vital for the sampling of particles that the pipes are made of conductive material, and electrically grounded. Otherwise, static charges may remove significant portions of the aerosol to be sampled. Short pieces of tubing might be replaced by conductive silicone tubing, which is elastic and conducting at the same time. A perfect inlet installation also avoids sources of turbulence (bends, connectors) as best as possible (turbulence enhances particle losses due to diffusion) and keeps the sampling lines as short as possible.

Figure 2 illustrates a custom-designed isokinetic flow splitter in which the sample flow velocity is near the flow velocity of the main flow. Another key feature of the splitter is that a sample is removed from the core of the main aerosol flow rather than from streamlines near the wall of the main pipe. This principle ensures a representative sampling especially of coarse and Nano-particles.

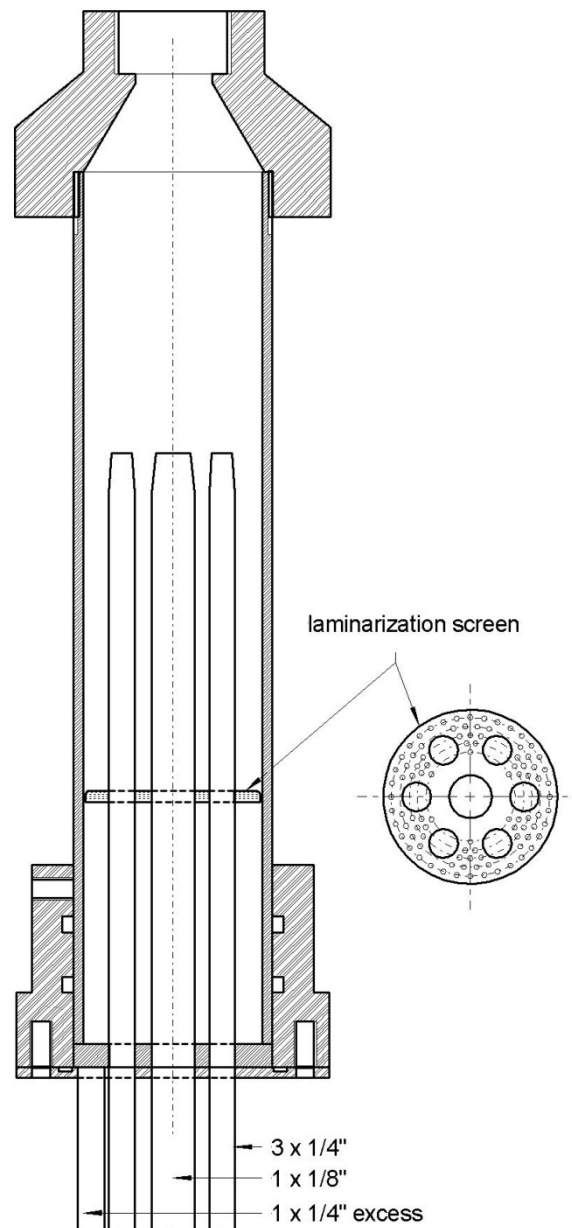


Figure 2: Example sketch for an isokinetic flow splitter

### General aspects of particle motion

The main challenge when transporting the aerosol to collectors and aerosol measuring instrumentation is to avoid particles losses. Particle loss mechanisms are size-dependent and are generally caused by particle diffusion, impaction, and sedimentation. Generally, losses due to particle diffusion are critical for ultrafine particles smaller than  $0.1 \mu\text{m}$ . In contrast, particle losses due to sedimentation and impaction are related to supermicrometer particles in horizontal and sloping pipes as well as bends. The configuration of the whole sampling configuration and the regime of the main air flow are strongly dependent on the purpose of the observational network.

The regime of an air flow in a pipe, laminar vs. turbulent, is characterized by its Reynolds Number (Re). A flow in a pipe is laminar up to a Reynolds Number of approximately 2000. Above this value, the flow becomes gradually more and more turbulent. The Reynolds Number of the flow can be determined by

$$\text{Re}_{\text{flow}} = \frac{\rho_G \cdot u_{\text{flow}} \cdot D_{\text{pipe}}}{\eta_G} \quad \text{Eq. 1}$$

Hereby is  $\rho_G$  the gas density,  $u_{\text{flow}}$  the flow velocity,  $D_{\text{pipe}}$  the inner diameter of the pipe, and  $\eta_G$  the gas viscosity.

The inertia of a particle in a flow is characterized by its Stokes Number Stk.

$$\text{Stk} = \frac{\tau \cdot u_{\text{flow}}}{D_{\text{pipe}}}$$

with

$$\tau = \frac{\rho_P \cdot D_P^2 \cdot C_C}{18\eta_G}$$

Hereby is  $\tau$  the relaxation time of the particle,  $u_{\text{flow}}$  the flow velocity,  $D_{\text{pipe}}$  the inner diameter of the pipe,  $\rho_P$  the particle density,  $D_P$  the particle diameter,  $C_C$  the Cunningham correction factor, and  $\eta_G$  the gas viscosity.

### Laminar flow sampling configuration

Generally, a laminar aerosol sampling is recommended in the ACTRIS network to minimize particle losses due to diffusion and inertia over a wide size range, especially for nucleation and coarse mode particles. Furthermore, the pressure drop from the inlet to the instruments can be kept in the range of few hPa. Minimum losses due to particle diffusion in a laminar flow can be achieved by keeping the length of the pipe as short as possible and the flow rate as high as possible. Particle losses of supermicrometer particles can be minimized by avoiding bends or horizontally orientated sampling pipes.

To design a laminar sampling configuration, the size-dependent particle penetration can be calculated (Hinds, 1992) by:

$$P = 1 - 5,5\mu^{2/3} + 3,77\mu$$

For  $\mu < 0.007$

$$P = 0.819 \cdot \exp(-11.5\mu) + 0.0975 \cdot \exp(-70.1\mu) + 0.0325 \cdot \exp(-179\mu)$$

For  $\mu > 0.007$

$$\mu = \frac{D \cdot L_{\text{pipe}}}{Q}$$

Hereby,  $D$  is the particle diffusion coefficient,  $L_{\text{pipe}}$  the length of the pipe, and  $Q$  the volume flow rate. In cases that bends cannot be avoided in the sampling pipe, the size-dependent particle penetration can be calculated by

$$P = 1 - Stk \cdot \frac{\theta^\circ}{180^\circ} \pi$$

Hereby,  $\theta$  is the angle of the bend.

Size-dependent losses due to sedimentation of supermicrometer particles in horizontal or sloping pipes can be calculated by

$$P = 1 - \frac{2}{\pi} \left[ 2\kappa \sqrt{1 - \kappa^{2/3}} - \kappa^{1/3} \sqrt{1 - \kappa^{2/3}} + \arcsin(\kappa^{1/3}) \right]$$

with

$$\begin{aligned} \kappa &= \varepsilon \cdot \sin(\theta) \\ \varepsilon &= \frac{3}{4} Z \\ Z &= \frac{L_{\text{pipe}}}{D_{\text{pipe}}} \cdot \frac{u_s}{\bar{u}_{\text{flow}}} \end{aligned}$$

Hereby,  $L_{\text{pipe}}$  is the length of the pipe,  $D_{\text{pipe}}$  the inner diameter of the pipe,  $u_s$  the sedimentation velocity,  $\bar{u}_{\text{flow}}$  the mean flow velocity, and  $\theta$  the angle of the pipe against the horizontal plain.